CPFD simulation of a dual fluidized bed cold flow model

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Abstract

The present work was carried out to simulate a cold flow model of a biomass gasification plant. The fluid dynamical behavior depends heavily on the particles' properties like the particle size distribution (PSD). For the simulation an Eulerian-Lagrangian approach, in particular by the multi-phase particle in cell (MP-PIC) method, was used to simulate particles with a defined PSD. Therefore, Barracuda VR, a software tool with an implemented MP-PIC method specifically designed for CPFD (computational particle fluid dynamics) simulations, was the software of choice. The simulation results were verified with data of previously conducted experiments on a physical cold flow model. The cold flow model was operated with air and bronze particles. The simulations were conducted with different drag laws: an energyminimization multi-scale (EMMS) approach, a blended Wen-Yu and Ergun (WYE) drag law, and a drag law of Ganser. Furthermore, a focus was set onto the normal particle stress (Ps value variation), which is significant in close-packed regions, and the loop seals' fluidization rate was varied to influence the particle circulation rate. The settings of the simulation were optimized, flooding behavior did not occur in advanced simulations, and the simulations reached a stable steady state behavior. The Ganser drag law combined with an adjusted Ps value with ($P_s = 30 Pa$) or without ($P_s = 50 Pa$) increased loop seal fluidization rates provided the best simulation results.

Keywords: Computational fluid dynamics, CPFD simulation, Cold flow model, Fluidized bed

1. Introduction

The demand for energy independency and renewable energy is increasing. Therefore, gasification technology is becoming more important. Dual fluid gasification is a promising technology to produce a product gas with a favorable gas composition which can be used for different products like synthetic natural gas and Fischer-Tropsch fuel.

Classical dual fluidized bed (DFB) gasification design in a bubbling bed without narrowings in the fuel reactor (FR) works well for conventional wood chips. There is

an increased interest from industry to utilize alternative low-cost fuels, but those fuels can lead to higher tar concentrations in the product gas stream. The design with narrowings in the FR column increases the solid gas interaction above the bubbling bed. This measure led to lower tar contents in the product gas and overall increased product gas quality [1].

CFD (computational fluid dynamics) is a diverse tool for applications in research and industry. The application of CFD in gassolid multi-phase systems does face new challenges compared to traditional simulations solely focusing on fluids. Therefore, Barracuda VR, a software tool with an implemented MP-PIC (multi-phase particle in cell) method specifically designed for CPFD (computational particle fluid dynamics) simulation, was the software of choice. Because of the high particle number, the MP-PIC approach is an efficient option for larger plants and applications.

CPFD simulations turned out to be very useful for the investigation of mixing behavior of fuel and bed material in dual fluidized bed plants. Optimal contact between fuel and bed material ensures low tar content which is favorable for long and stable operation of such plants. Better mixing can be achieved with additional fluidization agent as Kuba et al. have shown [2]. With CPFD simulations the effect of such additional fuel nozzles can be investigated and their impact on overall mixing process in the gasification or fuel reactor.

In the present work the CPFD simulation of a cold flow model used for the up-scaling and dimensioning of a biomass gasification plant is the object of interest. Cold flow models are an important tool to study fluid dynamical behavior, scale up small plants, and offer the possibility to optimize and test the design of a plant on a small scale.

A verified CPFD model can reflect trends and predict the process in a plant with a certain degree of accuracy as well as allow fast testing of different plant modifications and optimization.

The goal of this work was to create a working CPFD model in Barracuda which predicts the behavior of the real cold flow model accurately and to gain deeper understanding of the behavior of the DFB gasification as well as simulations of fluidized beds. The following questions were subject of interest:

- How do the different settings (e.g. (drag law, particle normal stress, etc.) in the CPFD software influence the particle behavior?
- What settings lead to the best performance at a specific operating point?
- Do the optimal settings predict reliable results at different operating points in the same system?

Barracuda has already been used for studies over a broad range of fluidization regimes, ranging from bubbling beds [3]–[5], spouting beds [6], risers [7]–[12], and full loop circulating fluidized beds (CFBs) [13]– [18]. A brief literature review of already conducted simulations of CFBs is given subsequently.

A full loop CFB was simulated by Clark et al. [13] with a Wen and Yu drag law approach. The flow behavior matched well with video recordings. The pressure drops were reasonable accurate.

Wang et al. [14] simulated a CFB to investigate the influence of various modeling parameters. They simulated the CFB with the Wen-Yu drag model, which predicted the particle circulation rate best, two configurations of the WYE drag model, which predicted the total pressure drop best, and the Ganser drag model, which overestimated both the pressure drop and particle circulation rate.

Hamilton et al. [15] simulated a CFB using the Wen-Yu drag model, which overpredicted the drag force in the simulation, with higher circulation rates but with good predictions of the trend.

Adkins et al. [16] simulated a CFB with the WYE and a Parker drag model. The WYE

drag model overestimated drag forces, while a modified drag correlation with an increased P_S value, derived by Parker, improved the correlation of the data.

Ma et al. [17] used the EMMS drag model to simulate a high-density CFB. The predicted pressure distribution deviated from experimental data in dense regions of the riser, while the particle circulation rate was predicted with relative errors less than 10%.

Kraft et al. [18] simulated a dual fluidized bed with EMMS, Ganser, Turton-Levenspiel, and WYE drag laws. The EMMS drag law predicted the pressure distribution best. while the others underestimated the pressure at the bottom. The best particle circulation rates were achieved with the Ganser drag law, while EMMS's were notable lower.

Based on the literature review the EMMS, Ganser, and WYE drag laws have been tested. Generally spoken the WYE and Ganser drag law cover both dense and dilute flow but seem to overpredict the drag force in some cases. The EMMS had a promising performance, predicted the pressure distribution well, but underpredicted the particle circulation rate in some cases.

1.1. Governing equations1.1.1. Equation for fluid phase

Continuity equation for the fluid without mass transport [19]:

$$\frac{\partial (\varepsilon_f \rho_f)}{\partial t} + \nabla \cdot (\varepsilon_f \rho_f \boldsymbol{u}_f) = \delta \dot{m}_p = 0$$

Eq. 1: Continuity equation

where ε_f is the fluid volume fraction, ρ_f is the fluid density, u_f the fluid velocity, and $\delta \dot{m}_p$ the gas mass production rate per volume from particle-gas chemistry. Momentum equation for the fluid with interphase momentum transfer function F[19]:

$$\frac{\partial (\varepsilon_f \rho_f \boldsymbol{u}_f)}{\partial t} + \nabla \cdot (\varepsilon_f \rho_f \boldsymbol{u}_f \boldsymbol{u}_f)$$

= $-\nabla p + \boldsymbol{F} + \varepsilon_f \rho_f \boldsymbol{g} + \nabla \cdot (\varepsilon_f \boldsymbol{\tau}_f)$

Eq. 2: Momentum equation

where p is the pressure, g the gravitational acceleration, and with the fluid stress tensor in index notation [19]:

$$\tau_{f,ij} = \mu \left(\frac{\partial u_{f,i}}{\partial x_j} + \frac{\partial u_{f,j}}{\partial x_i} \right) - \frac{2}{3} \mu \delta_{ij} \frac{\partial u_{f,k}}{\partial x_{f,k}}$$

Eq. 3: Stress tensor

where μ is the sum of laminar shear and turbulence viscosity, δ_{ij} is the Kronecker delta, $u_{f,i}$ the fluid's velocity, and x_i is the spatial variable.

1.1.2. Equations for particulate phase

The particle acceleration a_p , as described in Eq. 4, was extended with an additional term, the modified acceleration due to contact stress. The particle acceleration is calculated using a blended particle acceleration model developed by O'Rourke and Snider [20].

$$\boldsymbol{a}_{p} = D(\boldsymbol{u}_{f} - \boldsymbol{u}_{p}) - \frac{1}{\rho_{p}} \nabla p + \boldsymbol{g}$$
$$- \frac{1}{\varepsilon_{p} \rho_{p}} \nabla \tau_{p} + \frac{\overline{\boldsymbol{u}}_{p} - \boldsymbol{u}_{p}}{2 * \tau_{D}}$$



 $\overline{\boldsymbol{u}}_p$ is the particle mean velocity, the particle density ρ_p , and τ_D is a damping time due to inelastic particle collisions (see [20]).

The particle normal stress, τ_p , models the particle-particle interactions and is important near close-pack and has little effect elsewhere [21].

$$\tau_p = \frac{P_S * \varepsilon_p^{\beta}}{max[\varepsilon_{cp} - \varepsilon_p, \alpha(1 - \varepsilon_p)]}$$
Eq. 5: Particle normal stress

 ε_p is the particle volume fraction, ε_{cp} is the close-pack particle volume fraction, and P_S is a constant with the unit Pa. The constants

 α , a small number to avoid dividing by zero, and β are dimensionless.

The MP-PIC method used in Barracuda describes the dynamics of the particle phase using the particle probability distribution function $f(x, u_p, \rho_p, V_p, t)$ and the evolution of the particle phase is determined by solving the Liouville equation for f [22]:

$$\frac{\partial f}{\partial t} + \nabla \cdot (f \boldsymbol{u}_p) \nabla_{\boldsymbol{u}_p} \cdot (f \boldsymbol{a}_p) = 0$$

Eq. 6: Liouville equation for f

where ∇_{u_p} is the divergence operator with respect to particle velocity.

The particle volume fraction is related to f by [22]:

$$\varepsilon_p = \iint f \frac{m_p}{\rho_p} dm_p d\boldsymbol{u}_p$$

Eq. 7: Particle volume fraction

The interphase momentum transfer function per volume is defined as [22]:

$$\boldsymbol{F} = -\iint f\left(m_p \left[D(\boldsymbol{u}_f - \boldsymbol{u}_p) - \frac{\nabla p}{\rho_p}\right]\right) dm_p d\boldsymbol{u}_p$$

Eq. 8: Interphase momentum transfer

where *D* is the drag function.

1.2. Drag laws

In this paper the EMMS, Ganser, and WYE drag laws were used. All of those drag laws are already implemented and available in Barracuda. The EMMS drag law is based on energy-minimization multi-scale an approach and the drag coefficient is calculated from structure parameters of the solid interactions. For further gas information it is referred to Yang et al. [23]. The Ganser drag law is based on studies of Chhabra et al. [24] and Wen and Yu [25]. In this work the sphericity of the particles is 1. The WYE drag law was introduced by Gidaspow [26] and is a blend of the Wen and

Yu drag law, which is appropriate for more dilute systems and the Ergun equation [27], which is appropriate at higher packing fractions. For the formulas implemented in Barracuda for the drag laws it is referred to the Barracuda Virtual Reactor version 17.3.0 user manual [28] which can also be found in [29].

2. Concept and methodology

The simulations were based on an existing cold flow model located at TU Wien. Martinovic [30] conducted cold flow experiments on this model. Those results were used to verify and rate the performance of the simulation. A 3D model of the cold flow model was designed in Autodesk Inventor Professional 2018, as depicted in Fig. 1, and used for this work. The model has a total height of 1.57 m.

The main parts of the model are the air reactor (AR) column, the fuel reactor (FR) column, with four built-ins to narrow the cross-section, the FR bubbling bed (FR BB), a separator on top of the AR (AR SEP) and one on top of the FR column (FR SEP), and a total of three loop seals (Upper (ULS), Inner (ILS), and Lower loop seal (LLS)), which connect the separators with the AR and FR column respectively and the columns with each other.



Fig. 1: Cold flow model (left) [30], 3D model for simulation (right)

The diameter of the AR was 52 mm, the width of the square FR column 62 mm and 16 mm at the narrowing (as depicted in Fig. 1).

For the cold flow model investigations bronze particles as bed material and air as fluidization agent were used. During operation the bronze particles were carried up in the riser, separated from the fluid in the separator by gravity and transported through a loop seal into the FR column. Most of the particles traveled downwards, while forming smaller fluidized beds, to the FR bubbling bed. The upwards moving particles were recirculated over a separator and a loop seal to the FR column. In the bubbling bed the particles traveled through a loop seal back to the riser.

For the simulation several settings were tested in order to accurately predict the particle behavior in the cold flow model. A focus was put on the drag law, the particle normal stress properties and influence of the fluidization rate.

The best set-up was determined in a step-bystep approach.

The simulations were conducted using Barracuda VR version 17. The hardware infrastructure was a workstation with an Intel Core i5-3570, 16 GB RAM, and a GeForce TITAN X graphics card.

2.1. General settings

The general settings for the simulation are listed in Tab. 1. The PSD of the particles is depicted in Fig. 2.

Real cell number	483 338	
Total number of	$\sim 1.16*10^{10}$	
particles		
Number of	$\sim 2.19*10^{6}$	
numerical particles		
Particle density [30]	8800 kg/m ³	
Bulk density [30]	5100 kg/m ³	
Drag law	EMMS, WYE,	
Diag law	Ganser	
D	1 Pa (default),	
Ps	up to 100 Pa	
Turbulence model	LES	

Tab. 1: General settings overview



Fig. 2: Particle size distribution (PSD) of particles with a Sauter diameter of 81.7 μm [30]

2.2. Boundary conditions

In order to simulate a fluidizing bed, air had to be introduced and removed from the system. The air was able to exit the model at the pressure boundary conditions and was introduced into the system at the flow boundary conditions (see Fig. 3).

At each separator a pressure boundary condition with 1 atm was defined, permeable for air but not for particles.

In each loop seal one flow boundary condition, in the AR four (AR1_{lower}, AR1_{upper}, AR2_{lower}, AR2_{upper}) and in the bubbling bed of the FR two flow boundary conditions (FR1 and FR2) were defined. The flow boundary conditions were set as injection points ("jets") split up in several individual flow boundary conditions. The boundary conditions in the loop seal consisted of 4 jets each, the conditions in the FR of 5 each, the conditions in the AR of 8 each. Therefore, a total of 54 active flow boundary conditions were used in the simulation to inject air into the system.

The mass flow into the system was regularly set to 24 Nm³/h into the AR, 12 Nm³/h into the FR, and 0.6 to 0.8 Nm³/h in each loop seal (see Tab. 2). This mass flow was split up equally between the jets. An overview of the flow boundary conditions is given in Tab. 2.

Furthermore, the fluidization rate was altered in the loop seals in certain simulations to investigate its influence.

	No.	Position/	Vol.
	of	Height	flow
	jets	[mm]	$[Nm^3/h]$
FR1	5	889	2
FR2	5	1999	10
$AR1_{lower}$	8	111	3.69
$AR1_{upper}$	8	326	8.31
$AR2_{lower}$	8	425	8.31
AR2 _{upper}	8	601	3.69
LLS	4	11	0.8
ULS	4	1045	0.8
ILS	4	1045	0.6

Tab. 2: Flow conditions overview

2.3. Pressure measurement

The pressure was measured over time to evaluate the pressure profile, therefore, a total of 25 pressure measurement points were defined in the FR, AR, and LLS.

2.4. Evaluation

The performance of the simulation was rated qualitatively, by visual observation of the rendered animation of the simulation and the generated graphs (see Fig. 3). The performance was quantitatively rated based on the mass flow rate and the pressure profile including defined parameters. The defined parameters were the pressure profile parameter (P_{PP}) and the pressure gradient parameter (P_{PG}).

The P_{PP} value was defined as the pressure difference measured in the experiment and the simulation at the bottom of the FR column, which indicated the total amount of particles in the FR column. Therefore, the target value for P_{PP} ist zero.

$P_{PP} = p_{Simulation} - p_{Experiment}$ Eq. 9: Pressure profile parameter

The P_{PG} value was calculated by adding up the squared difference, between measured and simulated values, of the pressure gradient values in the FR column. Therefore, the target value for P_{PG} is zero and it indicated the quality of the distribution in the FR column.

$$P_{PG} = \sum_{i} \left(\left| p'(H_i)_{Simulation} - p'(H_i)_{Experiment} \right|^2 \right)$$

The circulation rate of the particles was measured with the mass flow of the particles in the AR column. If the simulation is in a steady state and no flooding occurred, this value will reflect the circulation rate of the particles. The measured value on the cold flow model in experiments was 823 kg/h (marked as a dashed line in Fig. 4, Fig. 5, and Fig. 6) with an AR fluidization rate of 24 Nm³/h.



Fig. 3: Visualization of the CPFD simulation (left), boundary conditions (BCs) for simulation (right)

3. Results and discussion

Overview of simulations

Abbre- viation	Drag Law	Mesh	PS [Pa]	LLS Fluidization rate [Nm ³ /h]	AR Fluidization rate [Nm ³ /h]
E300	EMMS	300k	1	0.8	24
G300	WYE	300k	1	0.8	24
W300	WYE	300k	1	0.8	24
G500	Ganser	500k	1	0.8	24
W500	WYE	500k	1	0.8	24
PS1	Ganser	500k	1	0.8	24
PS30	Ganser	500k	30	0.8	24
PS50	Ganser	500k	50	0.8	24
PS65	Ganser	500k	65	0.8	24
PS100	Ganser	500k	100	0.8	24
Gln	Ganser	500k	1	0.8	24
Gla	Ganser	500k	1	1.4	24
G30n	Ganser	500k	30	0.8	24
G30a	Ganser	500k	30	1.4	24
G30b	Ganser	500k	30	2	24
W1n	WYE	500k	1	0.8	24
W30n	WYE	500k	30	0.8	24
W30a	WYE	500k	30	1.4	24
E30n	EMMS	500k	30	0.8	24
E30a	EMMS	500k	30	1.4	24
16a	Ganser	500k	30	1.4	16
16b	Ganser	500k	50	0.8	16
20a	Ganser	500k	30	1.4	20
20b	Ganser	500k	50	0.8	20
24a	Ganser	500k	30	1.4	24
24b	Ganser	500k	50	0.8	24

Tab. 3: Overview of simulations

Drag law pre-selection

In order to find a suitable drag law for the simulation the three drag laws: Ganser, WYE, and EMMS were tested. First simulations were conducted with a coarser grid, with about 300 000 ("300k") cells (Fig. 4: "E300", "W300", "G300"), to investigate the general trend of the drag laws. Subsequent simulations were conducted as described previously with a mesh with approximately 500 000 ("500k") cells (Fig. 4: "G500" and "W500"). The defined parameters in Fig. 4 show a similar performance of both the Ganser and WYE drag laws, but with both drag laws and chosen settings flooding of the AR SEP was observed in the rendered animation. The AR SEP flooding occurred slower with the

Ganser drag law compared to the WYE drag law in the simulation with 500k cells.



Fig. 4: Drag law variation with 300k and 500k cells

The other parameters, P_{S} and LLS fluidization rate, were tested with all three drag laws to find the best settings. In general, the EMMS drag law did perform worse compared to the other drag laws. An explanation for those results could be found by looking at the conditions on which the implemented EMMS model ("EMMS-Yang-2004 model") in Barracuda was based on. Among other conditions, the model was generated for a particle density of 930 kg/m³ and a solid flux of 14.3 kg/m²s. The particle density of the bronze particles is 8800 kg/m³ and considering a diameter in the AR column of 52 mm, the mass flux is approx. $107 \text{ kg/m}^2\text{s}.$

P_S variation

In first simulations of the cold flow model it was observed from graphs and rendered animations, that the pressure drop and bed's height building up in the FR column are significantly higher than the measured values of the cold flow experiments. Additionally, it was a high particle volume fraction close to close-pack observed in the simulation. In order to reduce the number of particles in the FR column the P_s value was varied to influence the particle normal stress and particle acceleration (see Eq. 4). The P_s value variation was conducted from 1 Pa (default) to 100 Pa with the Ganser drag law. The simulation results are shown in Fig. 5.



Fig. 5: Ps constant variation with Ganser

The increased P_S value to 30 and 50 Pa improved the pressure drop and pressure gradient in the FR column (see Fig. 5) while improving the particle circulation rate.

Loop seal fluidization rate variation and different drag laws

The influence of the LLS fluidization rate on particle circulation rate was observed and if the LLS fluidization rate is limiting the particle circulation rate. If the LLS limits the particle circulation a small error would impact the whole simulation on the one hand. On the other hand, adjustments to the LLS fluidization rate would allow more realistic results by adjusting the model and knowing this possible error.



Fig. 6: Variation of loop seal fluidization rate -Simulations with non-increased loop seal fluidization rates are labeled as "n", increased by the factor 1.75 as "a", and increased by the factor 2.5 as "b"

The simulations with Ganser and an adjusted P_S value to 30 Pa reached a stable steady state (Fig. 6). They improved further with 1.75 times increased loop seal fluidization rates. The WYE drag law predicted an AR mass flow larger than the ULS mass flow rate in all tested cases, which led to flooding of the AR SEP and an unsteady state. The EMMS drag law underestimated the mass flows with all adjustments. Those trends reflected the behavior observed previously during drag law variation. Therefore, Ganser had the best performance for the simulations. The WYE and EMMS drag laws were tested further.

Different operating points

Different operating points of the cold flow model were tested, with set AR fluidization rates from 16 to 24 Nm³/h. The results are visualized in Fig. 7 and Fig. 8. The operating points were simulated with $P_S = 30$ Pa and altered loop seal fluidization rates, which are labeled with "a", and with $P_S = 50$ and nonaltered loop seal fluidization rates which are labeled as "b". The AR fluidization rate is indicated by the prefixed number in Fig. 7.



Fig. 7: Simulations of different operating point -Simulations with altered loop seal fluidization rates and $P_S = 30$ Pa are labeled as "a", with non-altered loop seal fluidization rates and $P_S = 50$ Pa are labeled as "b"



Fig. 8: Particle circulation rates at different operating points

The simulations with altered loop seal fluidization rates and Ps of 30 Pa predicted larger mass flow rates compared to the cases with normal loop seal fluidization rates and P_s of 50 Pa. The pressure difference over the FR column is in both cases very similar. The biggest was observed in particle distribution in the FR column represented as the pressure gradient. The pressure gradient deviation (P_{PG} value) visualized in Fig. 7 reflects the observation, that the cases with P_S set to 50 Pa are predicting the real particle distribution of the cold flow model better. Subsequently, the settings with P_S of 30 Pa and increased loop seal fluidization rates will be referred to as case "A", the settings with P_s of 50 Pa and normal loop seal fluidization rates will be referred to as case "B". The particle circulations rates of the simulation compared to the measured experimental circulation rates by Martinovic [30] are depicted in Fig. 8. The simulation with an AR fluidization rate of 16 Nm³/h overpredicted the particle circulation rate (302 kg/h) in case A by 26% and underpredicted in case B by 9%. The simulation with an AR fluidization rate of 20 Nm³/h overpredicted the particle circulation rate (504 kg/h) in case A by 19% and overpredicted in case B by 7%. The simulation with an AR fluidization rate of 24 Nm³/h overpredicted the particle circulation rate (823 kg/h) in case A by 1% and underpredicted in case B by 10%.

Overall the simulation with Ps set to 50 Pa and regular loop seal fluidization had a better performance due to the pressure parameters sufficient mass flow rates. and The simulations with Ps set to 30 Pa and increased loop seal fluidization rates tended to overpredict the mass flow rates and accumulated too many particles in the upper part of the FR column. The dependence of the particle circulation rate from the AR fluidization rate of the real case and the simulated cases is depicted in Fig. 8. Latter shows the clear trend of increasing particle with increased circulation rates AR fluidization Based rates. on those investigations, the best results on various operating points are calculated with a Ps value of 50 Pa.

4. Conclusion and outlook

The most important settings for the simulation are the drag law and the P_s constant. In particular the P_S constant played a significant role to achieve a steady state behavior, reduce bed heights in the FR column, and to achieve realistic particle circulation rates. The Ganser drag law and a Ps value of 50 Pa led to the best results overall. The loop seal fluidization rate had also a significant influence. Comparable results were achieved with a P_S value of 30 Pa and increased loop seal fluidization by 1.75, which led to a worse particle distribution but to a similar particle circulation rate. Different operating points were successfully simulated with those settings.

The ideal P_s value could be further investigated by repeating the simulation with slightly varied Ps values. Although good results were achieved by only adjusting the variation Ps value the of the constant could further improve the ß simulation and should be considered in future simulations. In the next stage the settings of the cold flow model could be used to simulate a pilot plant. After simulating the

cold flow operation of a pilot plant successfully, heat transfer and chemical reactions should be modeled and implemented. A reliable and robust CFD model allows faster and cost-effective testing. Different set-ups of the actual gasification plant can be tested without physical modifications to optimize the geometry. Furthermore, a computational simulation allows deeper insight into the plant itself and is not limited by sight or measuring points. Therefore. fluid dynamical behavior can be investigated at any desired location. This could lead to faster implementation of the technology and improved quality of the product gas with a reduced tar content.

5. List of abbreviations

abbreviation	meaning		
AR	air reactor		
BB	bubbling bed		
BCs	boundary conditions		
CFB	circulating fluidized bed		
CED	computational fluid		
CFD	dynamics		
CDED	computational particle		
	fluid dynamics		
DFB	dual fluidized bed		
e.g.	exempli gratia		
etc.	et cetera		
	energy-minimization		
EMMS	multi- scale (EMMS		
	drag model)		
Eq.	equation		
FR	fuel reactor		
IC	initial condition		
ILS	inner loop seal		
LES	large eddy simulation		
LLS	lower loop seal		
MP-PIC	multi phase particle in		
	cell		
PSD	particle size distribution		
SEP	separator		
ULS	upper loop seal		
WVE	Wen-Yu and Ergun		
W I L	(WYE drag model)		

symbol	meaning	SI unit
a_p	particle acceleration	$m_{/_{S^2}}$
D	drag function	$1/_{s}$
F	interphase momentum transfer function	N/m^3
р	pressure	Ра
g	gravitational acceleration	$m_{/_{S^2}}$
t	time	S
P_S	constant to calculate τ_p	Ра
\boldsymbol{u}_{f}	fluid velocity	$m_{/s}$
u_p	particle velocity	$m_{/s}$
$\overline{oldsymbol{u}}_p$	particle mean velocity	$m_{/s}$
<i>xi</i>	spatial variable	m
α	constant to calculate τ_p	-
β	constant to calculate τ_p	-
ε _{cp}	close-pack particle volume fraction	-
ε _f	fluid volume fraction	-
ε _p	particle volume fraction	-
δ_{ij}	Kronecker delta	-
m_p	particle mass	kg
μ	viscosity	kg_{m*s}
∇	nabla operator	1/m
ρ _f	fluid density	$\frac{kg}{m^3}$
τ_D	damping time due to inelastic particle collisions	S
τ_f	fluid stress tensor	N/m^2
τ _p	particle normal stress	N/m^2

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