Yu. Vetyukov

Motivation

Mathematical model

Finite elements

Simulations

Analytica solution

Conclusions

Dancing rod problem in the context of Lagrangian mechanics

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Configurational forces and nonlinear structural dynamics

C. Armanini, F. Dal Corso, D. Misseroni, D. Bigoni 🝳 🖾



- Flexible rod, low friction, concentrated mass
- Configurational force at the tip of the sleeve is related to energy release rate and prevents full injection.

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Mathematical model

Flexible rod partially sliding in a rigid sleeve

- Inextensible unshearable rod
- Distributed mass
- No friction
- Lagrangian (material) description inefficient
- Non-material mixed Eulerian-Lagrangian model
- Material coordinate s, length l
- Length of the free part $\eta(t)$
- Configurational force at the tip of the sleeve is proportional to local curvature and is work conjugate to η.





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Mathematical model Kinematic description Parametrization of free part:

 $\boldsymbol{x} = x\boldsymbol{e}_x + y\boldsymbol{e}_y = \boldsymbol{x}(\sigma, t),$

- $0 \le \sigma \le 1$
- Mapping:

$$s = l - \eta + \eta \sigma,$$

$$\sigma = 1 - (l - s)/\eta$$
(2)

• Strain energy requires 2nd order derivatives:

$$\partial_s \boldsymbol{x} = \partial_\sigma \boldsymbol{x} \, \partial_s \sigma = rac{1}{\eta} \partial_\sigma \boldsymbol{x},$$

 $\partial_s^2 \boldsymbol{x} = rac{1}{\eta^2} \partial_\sigma^2 \boldsymbol{x}$ (3)

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Mathematical model Kinematic description

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• Material velocity of a particle:

$$\dot{m{x}} = \partial_t m{x}|_{s= ext{const}} =$$

$$=\partial_t \boldsymbol{x}|_{\sigma= ext{const}}+\dot{\sigma}\,\partial_\sigma \boldsymbol{x}$$

$$\dot{s} = 0 \Rightarrow \dot{\sigma} = \frac{(1-\sigma)\dot{\eta}}{\eta}$$
(4)

• Boundary conditions at the tip of the sleeve:

$$x(0,t) = y(0,t) = 0,$$

 $\partial_{\sigma} y(0,t) = 0$ (5)

$$\eta$$
 g $\tilde{\alpha}$
 y $L-\eta$ H L



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Finite element approximation

- We divide the domain $0 \leq \sigma \leq 1$ into n finite elements.
- Generalized coordinates: nodal unknowns $m{x}_i,\,(\partial_\sigmam{x})_i$

• Cubic approximation on an element e

 $\begin{aligned} \boldsymbol{x}(\sigma) &= S_1(\sigma)\boldsymbol{x}_e + S_2(\sigma)(\partial_{\sigma}\boldsymbol{x})_e + S_3(\sigma)\boldsymbol{x}_{e+1} + S_4(\sigma)(\partial_{\sigma}\boldsymbol{x})_{e+1} \\ \text{(6)} \\ \text{guarantees } C^1 \text{ interelement continuity.} \end{aligned}$

- Further generalized coordinate is length of free part η.
- Axial strain

$$\varepsilon = \frac{1}{2} (\partial_s \boldsymbol{x} \cdot \partial_s \boldsymbol{x} - 1) \tag{7}$$

is penalized and $\varepsilon \to 0$ when $n \to \infty.$

Bending strain is curvature

$$\kappa = \partial_s^2 \boldsymbol{x} \cdot (\boldsymbol{e}_z \times \partial_s \boldsymbol{x}), \quad |\kappa| = |\partial_s^2 \boldsymbol{x}|.$$
 (8)

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Finite element approximation Energies and equations of motion

• Total energy of bending + penalty for axial strain:

$$U = \int_0^1 \frac{1}{2} (a\kappa^2 + b\varepsilon^2) \partial_\sigma s \,\mathrm{d}\sigma \tag{9}$$

Total potential of gravity:

$$W = -\int_{0}^{1} \rho g \boldsymbol{x} \cdot (\boldsymbol{e}_{x} \cos \alpha + \boldsymbol{e}_{y} \sin \alpha) \partial_{\sigma} s \, \mathrm{d}\sigma - \frac{1}{2} \rho g \cos \alpha (l - \eta)^{2}$$
(10)

Total kinetic energy:

$$T = \int_0^1 \frac{1}{2} \dot{\boldsymbol{x}} \cdot \dot{\boldsymbol{x}} \, \partial_\sigma s \, \mathrm{d}\sigma + \frac{1}{2} \rho (l - \eta) \dot{\eta}^2 \tag{11}$$

- Dissipation function R proportional to $\dot{\varepsilon}^2$ to damp out high frequency axial vibrations
- Lagrange's equations of motion for nodal d.o.f. and η

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Simulation results Very flexible rod with small initial injection

- Parameters (SI units):
 - Length l = 1, thickness $h = 0.5 \cdot 10^{-3}$
 - Material $E = 2 \cdot 10^{11}$, $\rho_3 = 7800$
 - Gravity g = 9.8, $\alpha = \pi/4$
 - Initial length of free part $\eta_0 = 0.7l$
- Results for n = 8 f.e. almost converged
- Length and tip deflection until full ejection at $t \approx 2.71$



Simulation validated against

Han, S.; Bauchau, O.A.: Configurational forces in variable-length beams for flexible multibody dynamics. Multibody System Dynamics, pp. 1-24, 2022.



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Very flexible rod with small initial injection

• Animation of the dynamic process

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Simulation results Less flexible rod

• Same process, thickness $h = 2 \cdot 10^{-3}$



- Seemingly periodic process
- Small vibration amplitude, first mode dominating

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Single term Ritz approximation at small vibrations

Assumptions

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- Small vibration amplitude
- First mode dominating
- Same axial motion for all particles
- Approximation with two generalized coordinates:

$$\boldsymbol{x}(\sigma,t) = \sigma \eta(t) \boldsymbol{e}_x + \gamma(t) w(\sigma) \boldsymbol{e}_y$$
 (12)

- Shape function $w(\sigma)$ is first vibration mode, w(1)=1
- Energy expressions follow by integration:

$$U = ak_U \gamma^2 / \eta^3,$$

$$W = \rho g (l(l - 2\eta) \cos \alpha - 2k_W \gamma \eta \sin \alpha) / 2,$$

$$T = \rho (\eta \dot{\gamma}^2 + \gamma \dot{\gamma} \dot{\eta} + 4(2k_T \gamma^2 + l\eta) \dot{\eta}^2 / \eta) / 8$$
th $k_U = 1.5453, k_T = 0.094385, k_W = 0.39150$
(13)



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Analytical solution Equations of motion and results

• Equations of motion take the form

$$\frac{1}{8}\rho\gamma\left(\ddot{\gamma} + \frac{32k_T\dot{\gamma}\dot{\eta}}{\eta} - 8gk_W\sin\alpha\right) + l\rho(\ddot{\eta} - g\cos\alpha) = \\
= \frac{\gamma^2(3ak_U + k_T\rho\eta^2(\dot{\eta}^2 - 2\eta\ddot{\eta}))}{\eta^4} \qquad (14)$$

$$\frac{2ak_U\gamma}{\eta^3} + \frac{1}{8}\rho\left(2\dot{\gamma}\dot{\eta} + 2\eta\ddot{\gamma} + \gamma\left(\ddot{\eta} - \frac{16k_T\dot{\eta}^2}{\eta}\right)\right) = \rho gk_W\eta\sin\alpha$$
• Results of time integration compared to f.e.



• Estimate for eigenfrequency $\omega = 2\sqrt{2ak_U/\rho}/\eta^2$ matches very well with numerical data.

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Conclusions & Outlook

- Non-material finite element model for large vibrations problem of a rod with solution dependent length of the free part is developed and validated.
- Configurational force results into alternating injection and ejection of the rod, even full ejection is possible.
- Semi-analytical two-d.o.f. model provides good results at small vibrations.
- Closed form estimates for characteristic values like maximal injection length are yet to be found.
- Elaborate investigation on the nature of the configurational force would allow taking friction at the tip of the sleeve into account.
- Further extension to the case of flexible sleeve necessary for approaching practically relevant formulations like concentric tube robots, used in the medicine.

Thank you for attention!